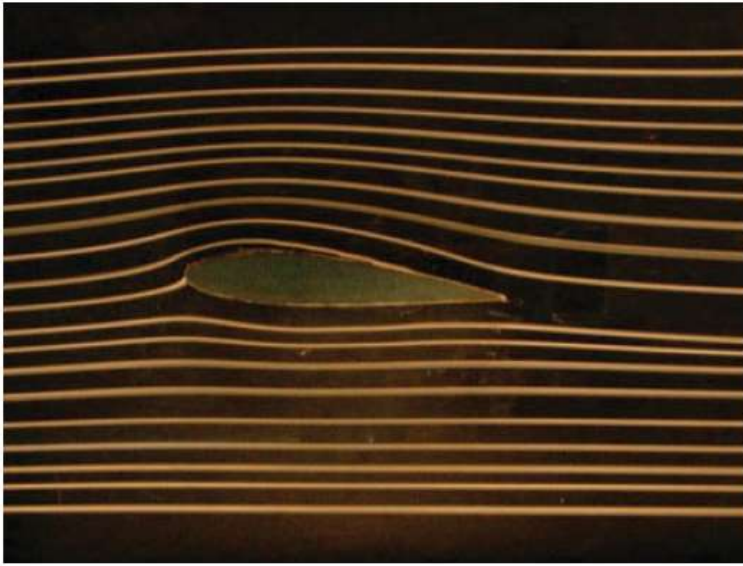


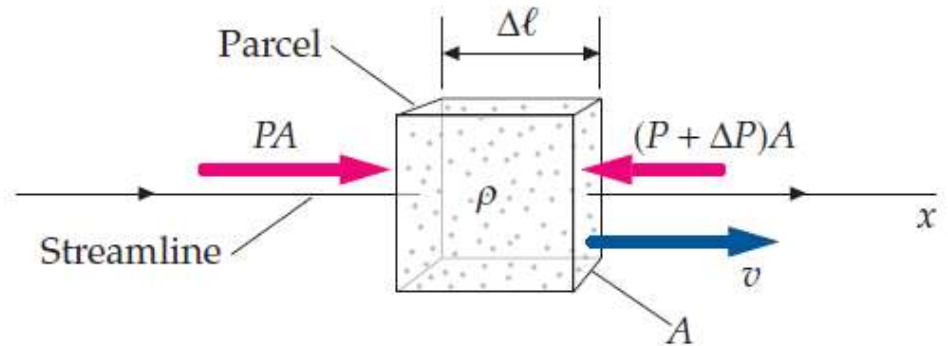
# Introduction to Physics I

For Biologists, Geoscientists, &  
Pharmaceutical Scientists

# Bernoulli equation

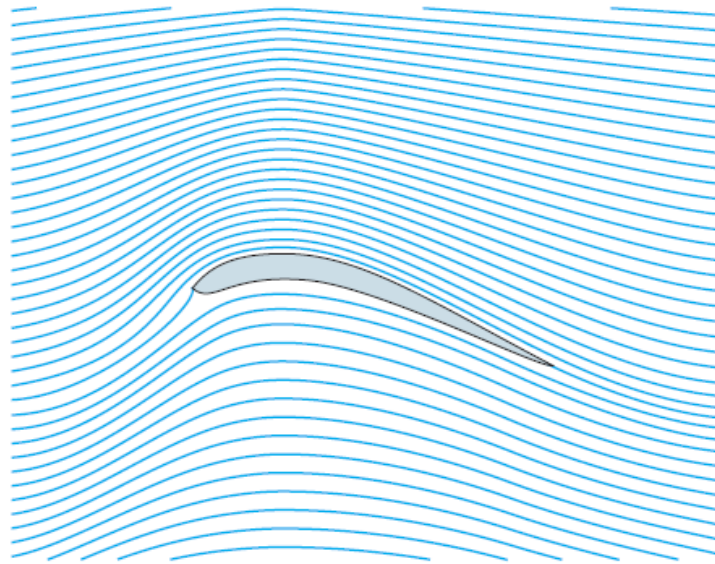


The streamlines are made visible by using smoke trails. In streamlined flow the particles of the fluid follow smoothly curved lines. (Holger Babinsky, 2003 *Phys. Educ.* 38 497-503.)



**FIGURE 13-17** The small parcel moves along a streamline into a region of reduced pressure.

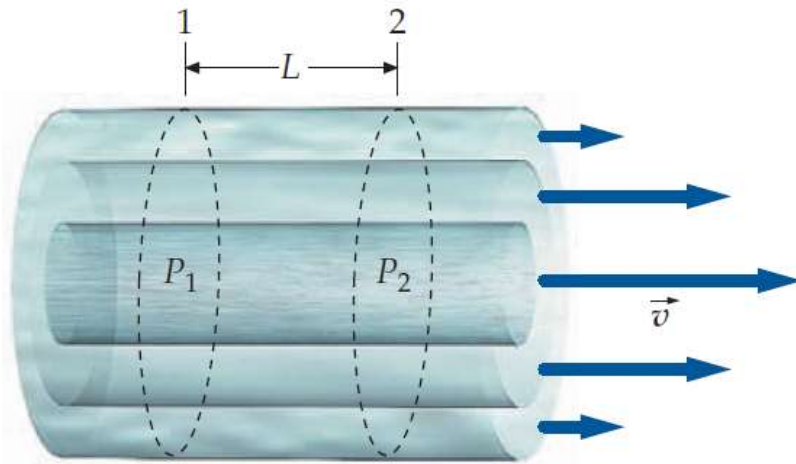
# wing and lift



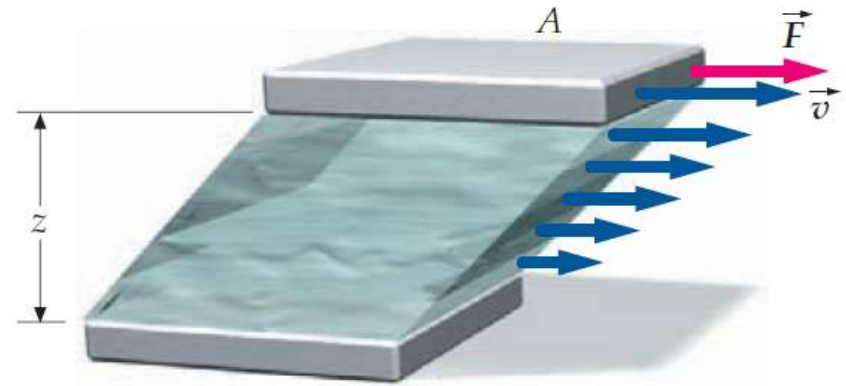
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**FIGURE 13-23** The purpose of an airfoil is to cause the streamlines to curve. Under normal conditions the streamlines will follow the curve of the airfoil. The airfoil shown is very thin, like the wing of a raptor. It is very efficient in creating lift.

# viscous fluid



**FIGURE 13-26** When a viscous fluid flows through a pipe, the speed is greatest at the center of the pipe. At the walls of the pipe, the speed of the fluid approaches zero.



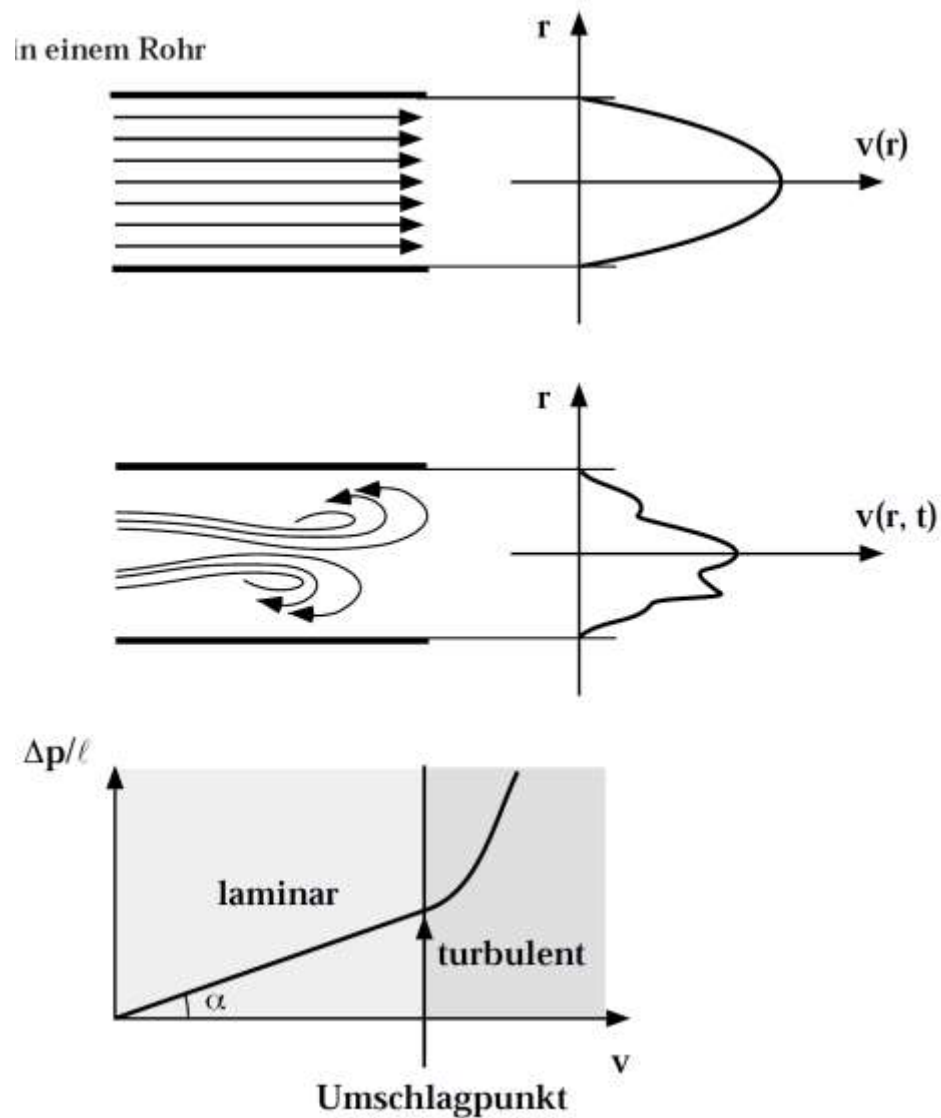
**FIGURE 13-27** Two plates of equal area with a viscous fluid between them. When the upper plate is moved relative to the lower one, each layer of fluid exerts a drag force on the adjacent layers. The force needed to pull the upper plate is directly proportional to  $v$  and the area  $A$ , and inversely proportional to  $z$ , the separation between the plates.

# viscous fluid

Tab. 5.4 Viskosität  $\eta$  einiger Flüssigkeiten und von Luft

Substanz (bei 20 °C)	Öle	Glyzerin	Blut ♂ Mittelwert	Blut ♀ Mittelwert	Äther	Hg	H <sub>2</sub> O	Luft
$\eta$ (Pa s)	1	0,83	0,0047	0,0044	0,0018	0,0015	0,001	$1,8 \cdot 10^{-5}$

# turbulent flow



# Biological systems & Reynolds nb

dynamical systems in aqueous environment at 310K  
constituted by small-size components

**Friction (Stokes)**

$$F_S = 6\pi\eta a v \sim 10^{-12} \text{N}$$

for a 1 $\mu\text{m}$  particle moving at 50 $\mu\text{m/s}$

**Stochastic (Brownian)**

$$F_B \propto \eta^2 / \rho \sim 10^{-9} \text{N}$$

$\eta$ : viscosity,  $10^{-3} \text{Ns/m}^2$

$\rho$ : density,  $10^3 \text{kg/m}^3$

essentially  
**no inertial forces**

# Living systems

- hydrodynamically speaking, we deal with fluids at low Reynolds number (*Purcell, 1976*)

$$R \cong vL\rho/\eta$$

$$\rho \sim 1 \text{ g/cm}^3, \eta \sim 1 \text{ g/(cm}\cdot\text{s)}$$

fish

$$v \sim 1 \text{ m/s}, L \sim 10 \text{ cm} \quad \Rightarrow$$

$$R \sim 10^5$$

*(accelerates water to propel itself)*

bacteria

$$v \sim 10 \text{ }\mu\text{m/s}, L \sim 1 \text{ }\mu\text{m} \quad \Rightarrow$$

$$R \sim 10^{-5}$$

*(uses viscous shear to move)*



# Living systems

$$Re = \frac{\text{inertial force}}{\text{viscous drag}}$$



Dolphin

$10^7$



„dolphin“

$10^5$

$10^3$



Jellyfish

$10$

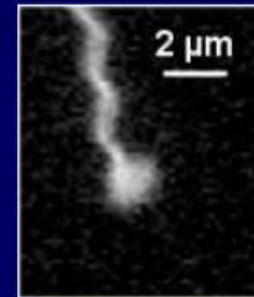
Macroscopic world ↑

Plankton



Micro-organisms ↓

Bacterium



$10^{-5}$   
no inertia !

# Living systems

⇒ bacteria thus know nothing about inertia

what's the distance it can coast when it stops swimming ?

$$m(-dv/dt) = 6\pi\eta av \quad \Rightarrow \quad v(t) = v(0) \cdot e^{-t/\tau}$$

with  $\tau = 2a^2\rho/9\eta \sim 2 \cdot 10^{-7} \text{ s}$

the bacteria thus stops in about a **micro-second** coasting a distance  $v(0) \cdot \tau \sim 0.04 \text{ \AA}$  (using  $v(0) = 20 \text{ \mu m/s}$ ).



*So bacteria swimming in water is like us swimming in honey*

## **Further reading**

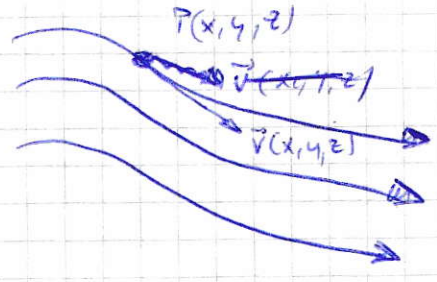
- E. M. Purcell, *Life at low Reynolds number* Am. J. Phys. 45, 3 (1977)
- H. C. Berg, *Motile Behavior of Bacteria* Physics Today, January (2000)

# hydrodynamics

slide 57

Euler (Leonhard), Bernoulli (David)

fluid in motion.



attribute at time  $t$  a density  $\rho(x, y, z)$  and velocity  $\vec{v}(x, y, z)$

$\forall P(x, y, z)$ , point in a liquid in motion

flow lines or streamlines

slides 58/59

defs: stationary / non-stationary, laminar / turbulent, etc.

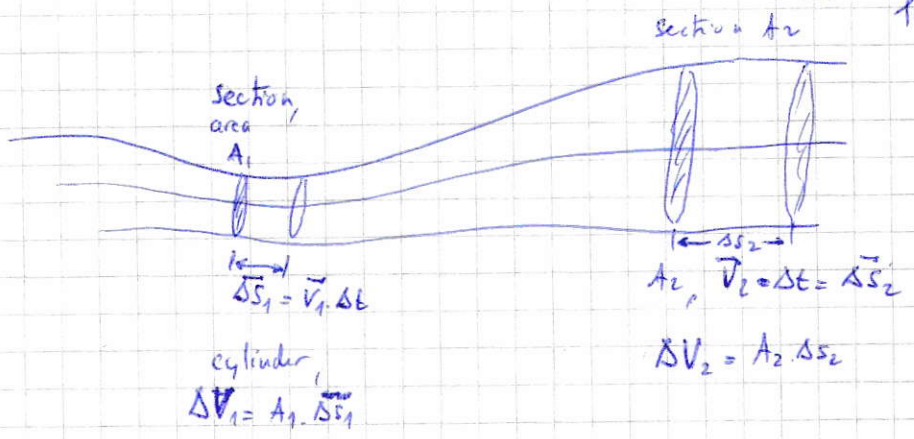
equation of continuity:

assumption (for later)

"ideal" liquid, i.e.:

- incompressible
- inviscid (no viscosity, or internal friction)

Consider tube with a tapered section: constant fluid flow (laminar)



mass flowing through area  $A_1$ :  $\Delta m_1 = \rho_1 \cdot \Delta V_1 = \rho_1 \cdot A_1 \cdot v_1 \cdot \Delta t$   
 ↑  
 density

mass flowing through area  $A_2$ :  $\Delta m_2 = \rho_2 \cdot \Delta V_2 = \rho_2 \cdot A_2 \cdot v_2 \cdot \Delta t$

no sources or sinks of fluids  $\Rightarrow \Delta m_1 = \Delta m_2$

or  $\parallel \rho \cdot A \cdot v = \text{const}$ , continuity equation

or  $\parallel A \cdot v = \text{const}$  for an incompressible medium ( $\rho = \text{const}$ )

Bernoulli equation

incompressible & inviscid liquid  
( $\rho = \text{const}$ ) (no viscosity)

Consider a parcel of fluid along a streamline laminar flow  
(show on slides)

slide

Fig 13-17 Tipler

air parcel (or cube) :  $m = \rho \cdot A \cdot \Delta L$ ; pressure difference  $\Delta P \Rightarrow$  force  $\vec{F}$   
(behind/abreast of parcel)

Newton :  $F = m \frac{dv}{dt} \approx m \cdot \frac{\Delta v}{\Delta t}$

force :  $F = P \cdot A - (P + \Delta P) \cdot A = -A \cdot \Delta P$

i.e. with Newton  $-A \cdot \Delta P = \rho \cdot A \cdot \Delta L \cdot \frac{\Delta v}{\Delta t}$   $\frac{\Delta L}{\Delta t} \approx v$

$\Delta P = -\rho \cdot v \cdot \Delta v$

for  $\Delta P \rightarrow 0$ ,  $dP = -\rho \cdot v \cdot dv$

integrate  $\int_{P_1}^{P_2} dP = -\rho \int_{v_1}^{v_2} v \cdot dv$ ,  $\rho = \text{const}$

$P_2 - P_1 = \frac{1}{2} \rho v_1^2 - \frac{1}{2} \rho v_2^2$

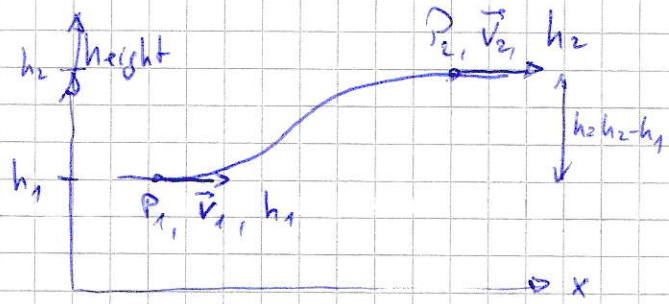
and

$P_1 + \frac{1}{2} \rho v_1^2 = P_2 + \frac{1}{2} \rho v_2^2$

and  $(v, P) \parallel P + \frac{1}{2} \rho v^2 = \text{const}$

If the streamline is not horizontal :

add gravitation term :  $g \cdot h$



i.e.  $\parallel P + \rho g \cdot h + \frac{1}{2} \rho v^2 = \text{const}$

Bernoulli eq.

$P + \rho g h$  represent a 'static' pressure

$\frac{1}{2} \rho v^2$  ; " " "dynamic" pressure

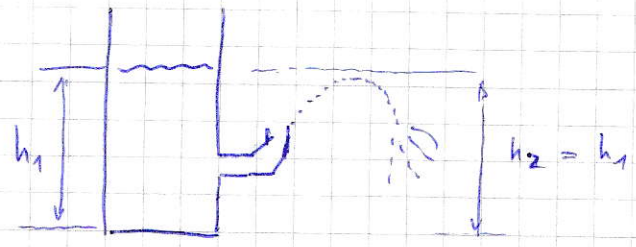
units:  $[P] = \frac{N}{m^2}$ , pressure

$[\frac{1}{2} \rho v^2] = \frac{kg \cdot m^2}{m^3 \cdot s^2} = \frac{kg}{m^2} \cdot \frac{m}{s^2} = \frac{1}{m^2} \cdot N = \frac{N}{m^2}$   
 $F = m \cdot dv/dt$

$[\rho g \cdot h] = \frac{kg}{m^3} \cdot \frac{m}{s^2} \cdot m = \frac{N}{m^2}$

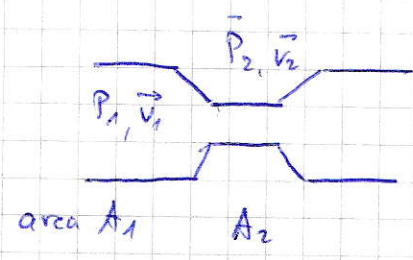
exp 107-12 Springbrunnen, fountain

$h_2 = h_1$   
 $h_1 \downarrow, h_2 \downarrow$



exp 107-13 Venturi effect

a) constriction in a pipe :

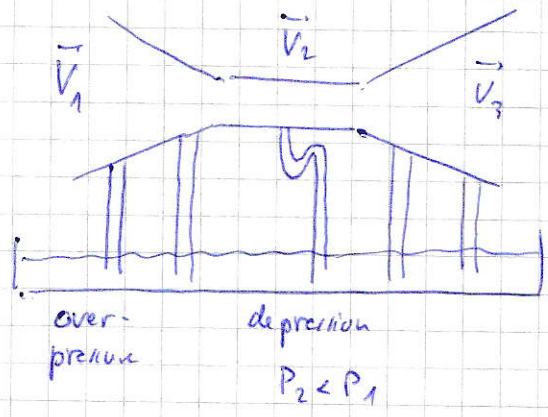


Bernoulli:  $P + \frac{1}{2} \rho \cdot v^2 = \text{const}$   
 eq. of continuity  
 $\Rightarrow v_2 > v_1$  (eq. of continuity)  
 $\rho = \text{const}$  as  $A_2 < A_1$   
 $A \cdot v = \text{const}$

and if  $v_2 > v_1 \Rightarrow P_2 < P_1$   
 Bernoulli

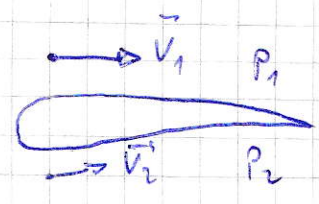
b) show Venturi tube

$\Rightarrow$  if geometry known ( $A_1, A_2$ )  
 and density known,  
 Venturi tube to measure  
 fluid velocity



Creating lift using a wing

slide wing : airfoil forces streamlines to curve and  $v_1 > v_2$



Bernoulli

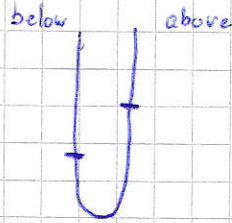
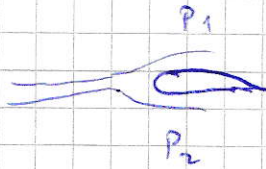
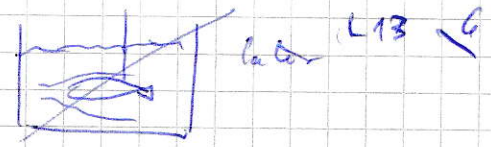
$P_1 + \frac{1}{2} \rho v_1^2 = P_2 + \frac{1}{2} \rho v_2^2$   
 $P_1 - P_2 = \frac{1}{2} \rho v_2^2 - \frac{1}{2} \rho v_1^2$   
 $v_1 > v_2 \Rightarrow P_1 < P_2$ , lift (upward force)

NB: air is compressible, hence Bernoulli cannot be strictly applied; and air pressure change due to gravity neglected

exp  
107-14

a) ~~Stromung / Apparat~~  
Flugzeugflügel

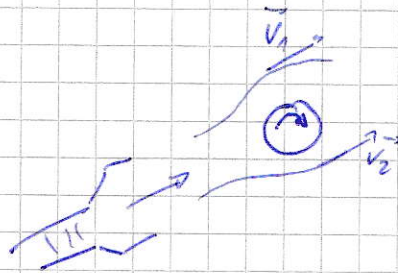
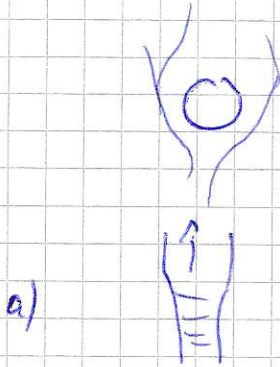
wing and pressure difference meas.



→ change angle to emphasize effect.

exp

ball in air flow



$v_1 > v_2$ , friction at air-ball interface  
 $F_1 \neq F_2$ , rotation  
fast flow  
⇒ low p (compared to atmosphere)  
⇒ "stick" to outlet

corollary: rotating ball with horizontal ~~fast~~ speed as a start (football) will/can gain height due to rotation effect and the resulting different flow pres. above and below

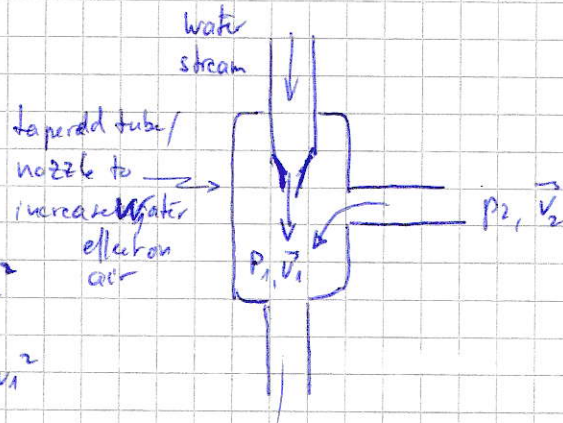
exp  
107-15

water tap pump

Bernoulli:

$$P_1 + \frac{1}{2} \rho \cdot v_1^2 = P_2 + \frac{1}{2} \rho \cdot v_2^2$$

$$P_1 - P_2 = \frac{1}{2} \rho v_2^2 - \frac{1}{2} \rho v_1^2$$



Viscous flow

(slide)  
viscous flow

Bernoulli:  $P + \frac{1}{2} \rho v^2 = \text{const}$ , horizontal tube

laminar flow:  $v = \text{const} \Rightarrow P = \text{const}$   
 $\rho = \text{const}$

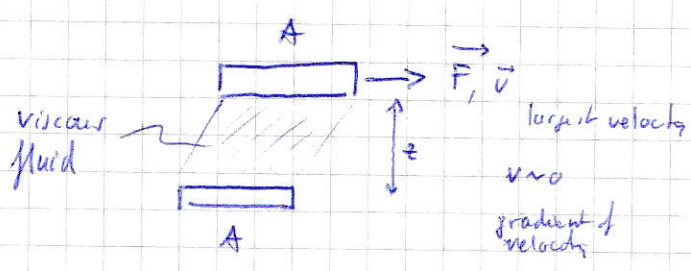
BUT: exp show  $P$  decreases downstream

(slide)

viscous flow  $\Delta P = P_1 - P_2 \propto v \cdot A$ , flow rate  
picture: concentric cylinders  
 $\uparrow$  area  
 $\uparrow$  speed

def viscosity, parallel plates

(slide)



def:  $\parallel F = \eta \frac{v \cdot A}{z}$ ,  $\eta$ : viscosity (dynamic) (prop. constant)

$[\eta] = \frac{N \cdot s}{m^2} = Pa \cdot s$   
 $\uparrow$  Pascal

or better

$\parallel F = \eta \cdot A \cdot \frac{dv}{dz}$   
 $\uparrow$  velocity gradient

resistance to flow:  $R$

start exp

Hagen-Poiseuille (1841)

seen above:  $\parallel \Delta P = P_1 - P_2 \propto v \cdot A$ , volume flow rate

$\equiv R \cdot v \cdot A$   
def  $\uparrow$  resistance to flow  $\parallel R = \frac{\Delta P}{v \cdot A} \propto \Delta P$  % unit

Poiseuille law (see demo below):  $R = \frac{8\eta \cdot L}{\pi \cdot r^4}$  for a tube with radius  $r$  length  $L$

and

$\parallel \Delta P = \frac{8\eta L}{\pi \cdot r^4} \cdot v \cdot A$ ,  $\propto \frac{1}{r^4}$

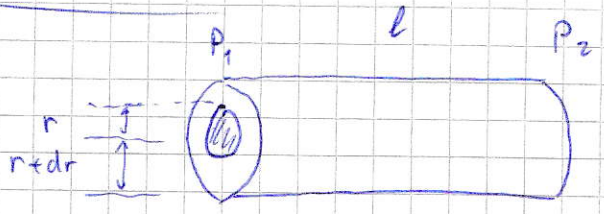
for a viscous flow  $v \cdot A = I_v \propto r^4$   
volume flow rate

NB: fluid is said to be "Newtonian" if  $\eta$  is indep. of  $\frac{dv}{dz}$

(slide)

typ. viscosity values,

demo Poiseuille



tube, radius  $R$   
(concentric cylinders)

per def viscosity

$$F_f = \eta \cdot A \cdot \frac{dv}{dr}$$

friction force

with  $A = 2\pi r \cdot l$ , contact surface between cylinders

force expressed via  $\Delta P$ :  $F_p = \Delta P \cdot S = (P_1 - P_2) \cdot \pi \cdot r^2$

Equilibr:  $F_R + F_p = 0$

$\Rightarrow 2\pi r \cdot l \cdot \eta \cdot \frac{dv}{dr} = -\Delta P \cdot \pi r^2$

$$\frac{dv}{dr} = -\frac{\Delta P}{2\eta l} \cdot r$$

$$\int_0^r dv = -\frac{\Delta P}{2\eta l} \int_0^r r \cdot dr$$

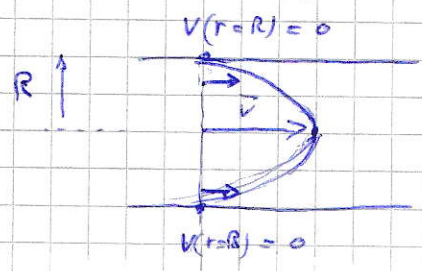
$$v(r) - v(0) = -\frac{\Delta P}{4\eta l} \cdot r^2$$

$v(r) = v(0) - \frac{\Delta P}{4\eta l} \cdot r^2$   
parabolic velocity distribution

At  $r = R$ ,  $v = 0$ : no motion at interface tube / liquid (of schematic)

$$v(R) = v(0) - \frac{\Delta P}{4\eta l} \cdot R^2 = 0$$

and  $v(0) = \frac{\Delta P}{4\eta l} \cdot R^2$



Volume flow rate  $I_v = \int_0^R v(r) \cdot dA$

rem.  $I_v = v \cdot A$

$$dA = 2\pi r \cdot dr \Rightarrow \int_0^R \left( v(0) - \frac{\Delta P}{4\eta l} \cdot r^2 \right) \cdot 2\pi r \cdot dr$$

$$= \frac{2\pi R^2}{2} v(0) - \frac{2\pi \cdot \Delta P \cdot R^4}{4\eta l \cdot 4}$$

$$I_v = \frac{\pi \cdot R^2 \cdot \Delta P \cdot R^2}{4\eta l} - \frac{\pi \cdot \Delta P \cdot R^4}{2\eta l \cdot 2} = \frac{\pi \cdot R^4}{8 \cdot \eta \cdot l} \cdot \Delta P$$

flow rate

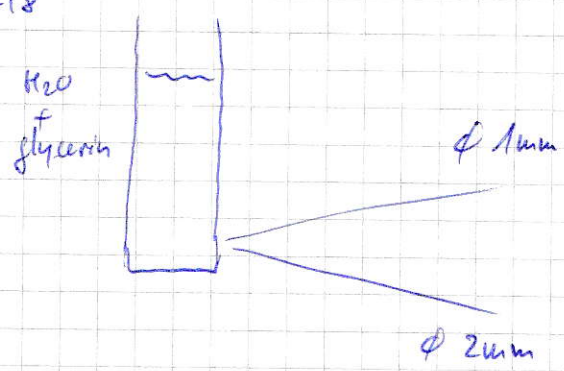
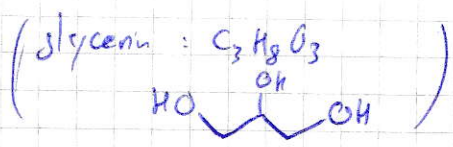
$$v(0) = \frac{\Delta P \cdot R^2}{4\eta l}$$

ged



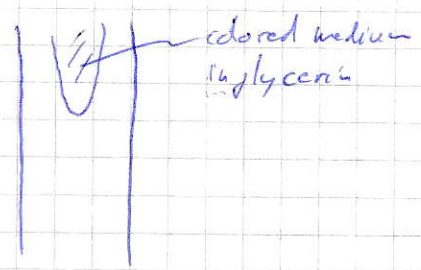
exp  
107-18

Hagen-Poiseuille



$\Delta v \propto R^4$

exp parabolic speed distribution

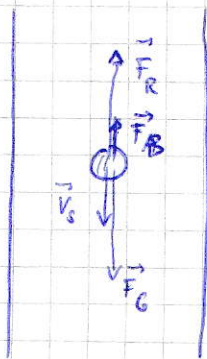


Stokes friction & sedimentation  
(drag force)

friction/drag force for a body moving in viscous liquid at velocity  $v$  ( $v$  not too large)

$\vec{F}_R = -6\pi\eta R \cdot \vec{v}$ ,  $R$ : radius object  
 Stokes,  $F_R \propto v$  for  $v$  small

sedimentation:



$\vec{F}_G + \vec{F}_B + \vec{F}_R = 0$   
 gravitation  $\uparrow$  drag  
 buoyancy (Archimedes)  
 volume sphere

$mg - \frac{m}{S_o} \cdot S_f \cdot g - 6\pi\eta R \cdot v_s = 0$

density  
 $S_o$ : object  
 $S_f$ : fluid

hence  $v_s = \frac{1}{6\pi\eta R} \cdot mg \left(1 - \frac{S_f}{S_o}\right)$ ,  $m = \frac{4\pi}{3} R^3 \cdot S_o$

$\parallel v_s = \frac{2}{9\eta} \cdot (S_o - S_f) \cdot g \cdot R^2$ ,  $\propto R^2$

$\uparrow$  centrifuge to collect small objects (DNA, cells, ...)

exp sedimentation: 3 different  $\phi$   
 107-22

# turbulent flow

L13 ✓

when  $v \nearrow$ , crossover laminar to turbulent

slide

critical speed at which crossover takes place

$$v_R = \frac{\eta}{\rho \cdot R} \cdot N_R$$

R: radius, tube

$N_R$ : const.

when  $v \nearrow$ , and  $\frac{\Delta p}{l}$ , pressure drop (viscous fluid) per unit length  $\nearrow$ ,  $\rightarrow$  turbulent

exp 1  
107-23

turbulent flow: colored medi- in tube

a) low  $v$ : laminar

b)  $v \nearrow$ , crossover to turbulent

NB: liquid mixing: turbulent regime!

exp:

stomach model: crossover / change laminar turbulent  
object shape

a) slow wing, edge angle

• Reynolds number  $N_R$

$$\| N_R = \frac{\rho \cdot v \cdot d}{\eta}$$

d: typical dimension, characteristic length of object

$N_R > 10^3 \rightarrow$  turbulent

slides

life at low Reynolds nb